

# HW04

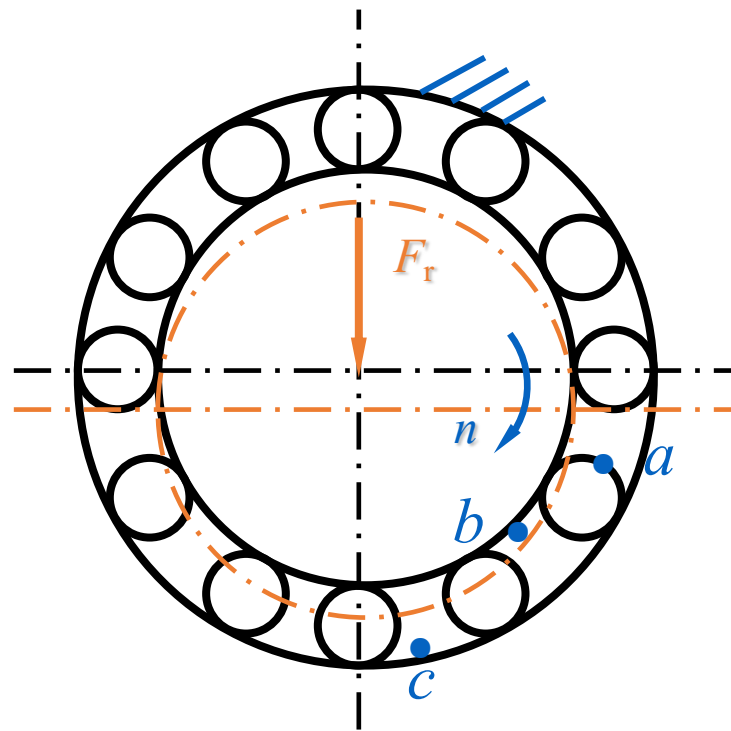
# 第04章 轴承及轴设计 作业

所有作业要求手写

Autumn 2024

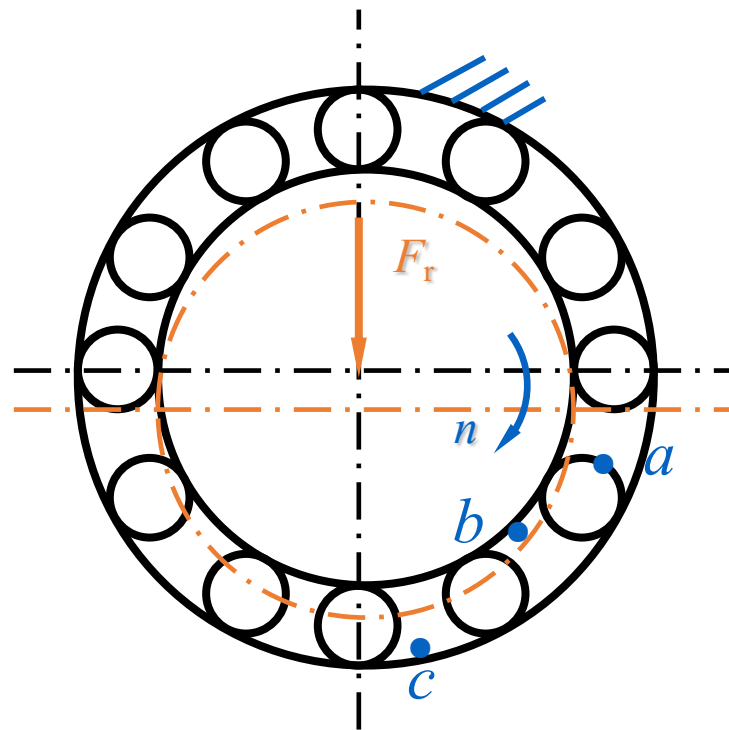
## HW 04.1

- 如右图所示，当轴承仅受到纯径向力作用时，对于滚动体上一点 a、内圈滚道接触点 b，以及外圈滚道接触点 c，请分别分析 a、b、c 三点的受力情况，并画出接触应力变化简图。



# HW 04.1

- As shown in the figure on the right, when the bearing is only subjected to a pure radial force, analyze the force conditions at points  $a$ ,  $b$ , and  $c$  on the rolling element, the inner ring raceway contact point, and the outer ring raceway contact point, respectively. Draw a simplified diagram of the contact stress variations



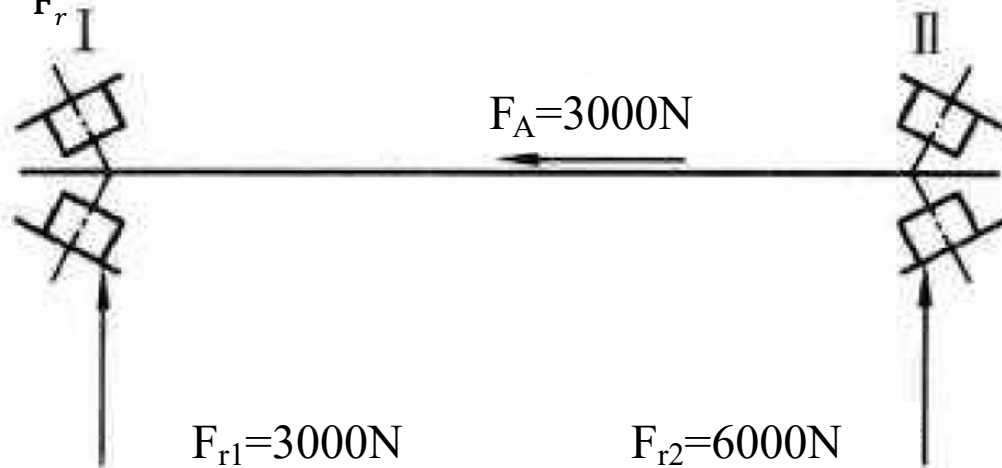
# HW 04.2

- 某设备中的一转轴，两端用 30207E 型轴承（如图所示）。轴工作转速  $n=1450 \text{ r/min}$ ，在常温下工作  $f_1=1$ ，轴所受轴向载荷  $F_A=3000\text{N}$ ，轴承所受的径向负荷  $F_{r1}=3000\text{N}$ ， $F_{r2}=6000\text{N}$ ，设计寿命  $L_h=1500\text{h}$ ，负荷系数  $f_p=1.5$ 。

(1) 求轴承派生轴向力  $S_1$ 、 $S_2$  的大小和方向

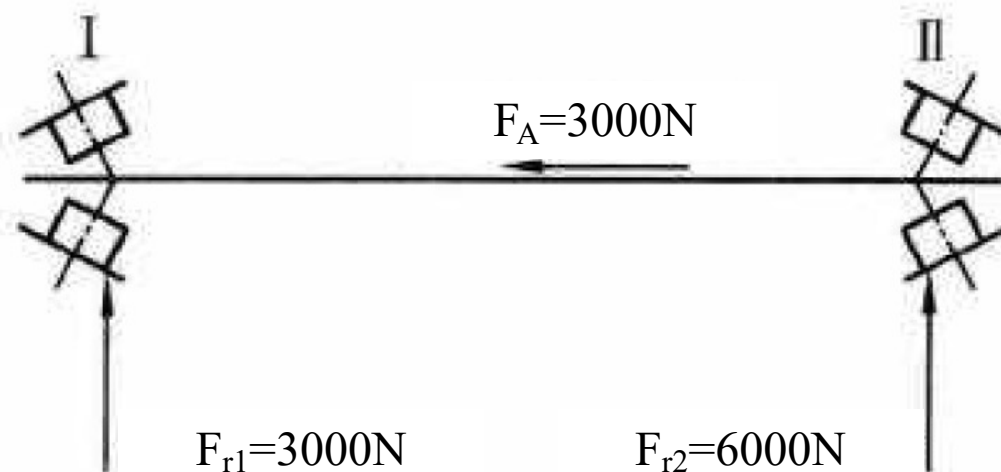
(2) 试校核该轴承是否满足寿命要求？

附：30207E 型轴承有关参数如下： $C_r=51500\text{N}$ ， $e=0.37$ ， $S=\frac{F_r}{2Y}$ ，当  $\frac{F_a}{F_r} \leq e$  时， $X=1$ ， $Y=0$ ；当  $\frac{F_a}{F_r} > e$  时， $X=0.4$ ， $Y=1.6$ 。寿命计算式： $L_h = \frac{10^6}{60n} \left(\frac{C_r}{P_r}\right)^\varepsilon$  ( $\varepsilon=10/3$ )



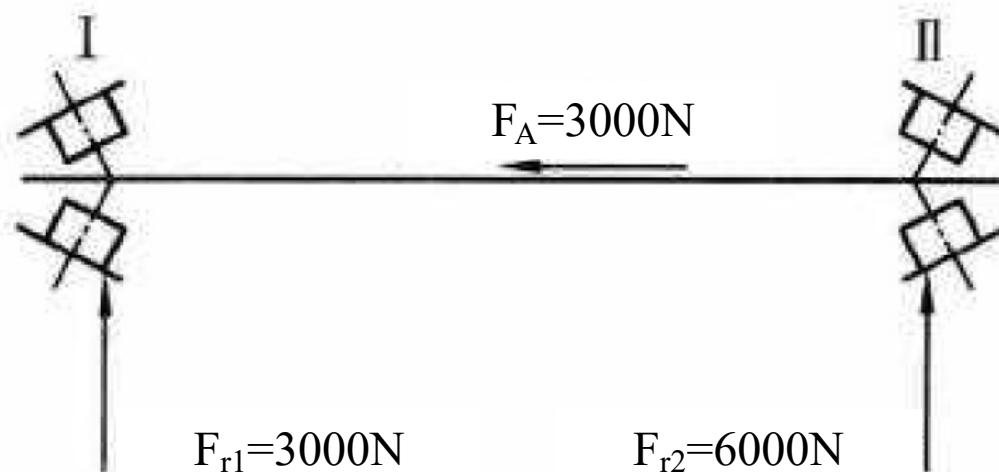
## HW 04.2

- A certain shaft in a piece of equipment is supported at both ends by 30207E-type bearings (as shown in the figure). The shaft operates at a speed of  $n=1450$  r/min, works at normal temperature with  $f_1=1$ , and is subjected to an axial load of  $F_A=3000$ N. The radial loads on the bearings are  $F_{r1}=3000$ N and  $F_{r2}=6000$ N. The designed service life is  $L_h=1500$  hours, and the load factor is  $f_p=1.5$ .



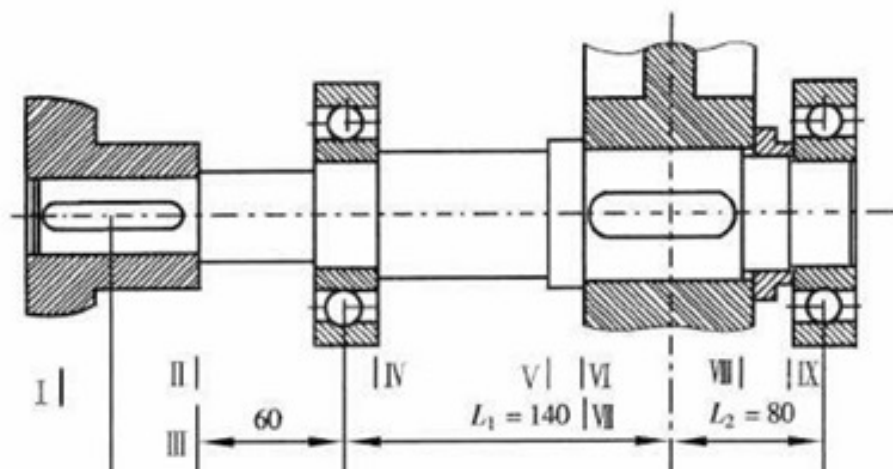
# HW 04.2

- (1) Determine the magnitude and direction of the axial forces S1 and S2 derived from the bearing.
- (2) Please check whether the bearing meets the service life requirements.
  - Appendix: The relevant parameters for the 30207E type bearing are as follows:
  - $C_r=51500\text{N}$ ,  $e=0.37$ ,  $S=\frac{F_r}{2Y}$ . When  $\frac{F_a}{F_r} \leq e$ ,  $X=1$ ,  $Y=0$ ; When  $\frac{F_a}{F_r} > e$ ,  $X=0.4$ ,  $Y=1.6$ .
  - Service life calculation formula:  $L_h = \frac{10^6}{60n} \left(\frac{C_r}{P_r}\right)^\varepsilon$  ( $\varepsilon=10/3$ )



# HW 04.3

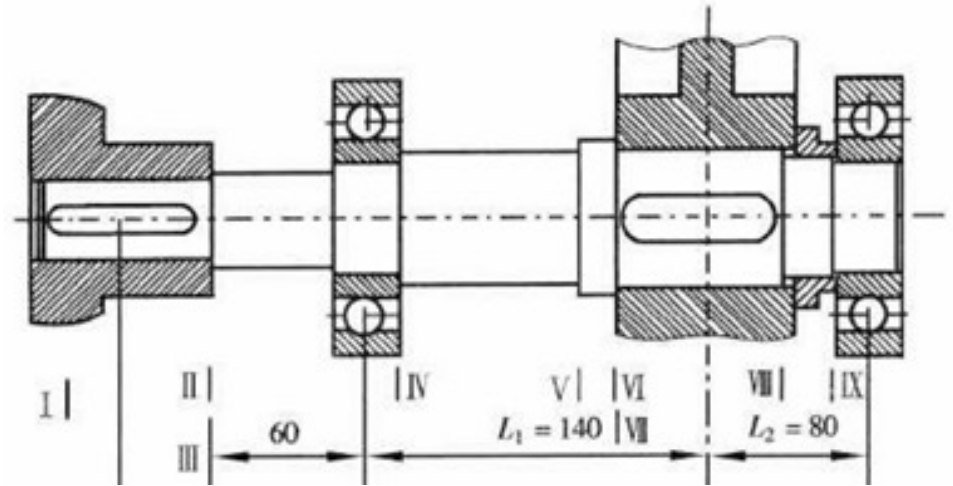
两级标准圆柱齿轮减速器输出轴的结构如图所示。已知齿轮分度圆直径  $d = 332 \text{ mm}$ ，作用在齿轮上的圆周力  $F_t = 7780 \text{ N}$ ，径向力  $F_r = 2860 \text{ N}$ ，轴向力  $F_n = 1100 \text{ N}$ ，单向工作。支点与齿轮中点的距离  $L_1 = 140 \text{ mm}$ ， $L_2 = 80 \text{ mm}$ 。



- 画出轴的受力简图；
- 计算支承反力；
- 画出轴的弯矩图、合成弯矩图及转矩图；
- 指出危险剖面的位置。

# HW 04.3

The structure of the output shaft of a two-stage standard cylindrical gear reducer is shown in the Figure. It is known that the diameter of the gear indexing circle  $d = 332$  mm, the circumferential force  $F_t = 7780$  N, the radial force  $F_r = 2860$  N, and the axial force  $F_a = 1100$  N, which is working in one direction. The distance between the pivot point and the midpoint of the gear  $L_1 = 140$  mm,  $L_2 = 80$  mm.



- Draw a force sketch of the shaft;
- Calculate the support reaction force;
- Draw the bending moment diagram, synthetic bending moment diagram and torque diagram of the shaft;
- Indicate the location of the hazardous profile.





机械设计

Mechanical Design

**Thank you~**

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