



ME311 | 机械设计

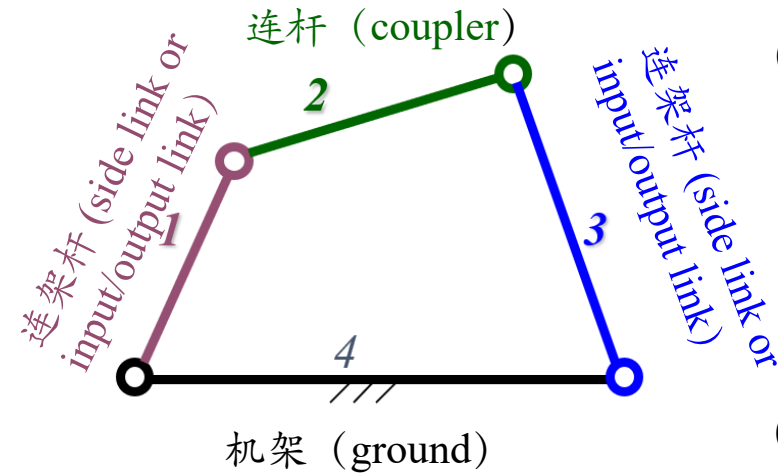
2023年秋季

# HW02

## 第02章 平面连杆机构 答案

南方科技大学

## HW 02.1



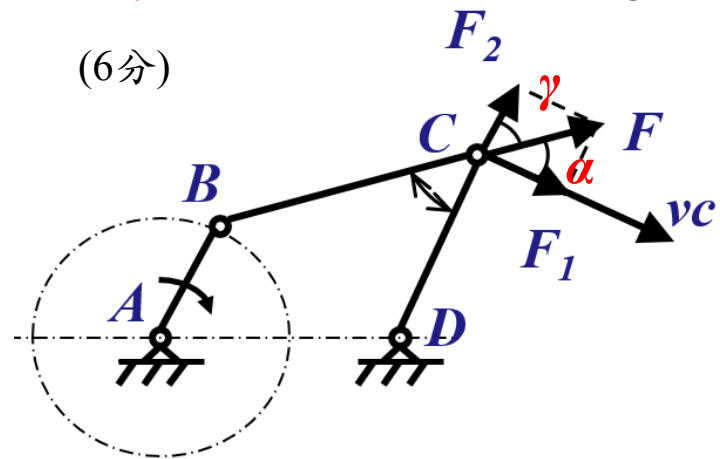
(1) 在**连架杆**中，能绕其轴线**回转360°**者称为曲柄；仅能绕其轴线**往复摆动**者称为摇杆。（红色部分每个关键点2分） A **side link** that can **perform a complete revolution** is called a crank; a side link that **cannot perform a complete revolute** is called a rocker.

(2) (i) 双曲柄机构:  $S+L \leq P+Q$ , 且最短杆为机架。  
 (ii) 曲柄摇杆机构:  $S+L \leq P+Q$ , 且最短杆的相邻构件为机架。其中最短杆为曲柄, 另一连架杆为摇杆;  
 (iii) 双摇杆机构:  $S+L \leq P+Q$ , 且最短杆的对边构件为机架。或者  $S+L > P+Q$ , 且  $L < S+P+Q$ .

(i) Double crank:  $S+L \leq P+Q$  (Grashof condition), and the shortest link is the ground. (2分)  
 (ii) Crank rocker:  $S+L \leq P+Q$ , and the shortest link is a side link. In this case, the shortest link is a crank and the other side link is a rocker. (2分)  
 (iii) Double rocker:  $S+L \leq P+Q$ , and the shortest link is the coupler. Or  $S+L > P+Q$ , while  $L < S+P+Q$ . (4分)

(3)  $\alpha$ : 压力角 pressure angle  
 $\gamma$ : 传动角 transmission angle

(6分)



# HW02.2

1、平面四杆机构中，是否存在死点，取决于\_\_B\_\_是否与连杆共线。

A. 主动件      B. 从动件      C. 机架      D. 摇杆

2、一个K大于1的铰链四杆机构与K=1的对心曲柄滑块机构串联组合，该串联组合而成的机构的行程变化系数K\_\_A\_\_。

A. 大于1      B. 小于1      C. 等于1      D. 等于2

3、在设计铰链四杆机构时，应使最小传动角 $\gamma_{\min}$ \_\_B\_\_。

A. 尽可能小一些      B. 尽可能大一些      C. 为 $0^\circ$       D.  $45^\circ$

4、平面连杆机构是由许多刚性体由\_\_C\_\_联结而成的机构。

A. 转动副      B. 高副      C. 低副

# HW02.2

1. For a planar four bar linkage, the existence of a dead center is depends on whether the \_\_\_B\_\_\_ is collinear with the coupler.  
A. Input link      B. Output link      C. Ground      D. Rocker
2. A planar 4R linkage ( $K > 1$ ) is combined in series with a centric slider-crank mechanism ( $K = 1$ ), then the coefficient of travel speed variation ( $K$ ) of the new mechanism is \_\_\_A\_\_\_.  
A. Bigger than 1      B. Smaller than 1      C. 1      D. 2
3. When designing a 4R linkage, the minimum transmission angle,  $\gamma_{\min}$ , should be \_\_\_B\_\_\_.  
A. As small as possible      B. As big as possible      C.  $0^\circ$       D.  $45^\circ$
4. A planar 4R linkage connects the rigid links by \_\_\_C\_\_\_.  
A. Revolute joints      B. higher pairs      C. lower pairs

# HW 02.3

(1) 若 AD 为最长杆, 则

$$L_{CD} + L_{AD} \leq L_{BC} + L_{AB}, \quad L_{AD} \geq L_{BC} \quad (5分)$$

即

$$200 + L_{AD} \leq 300 + 250, \quad L_{AD} \leq 350 \quad (3分)$$

即

$$300 \leq L_{AD} \leq 350 \quad (2分)$$

(1) If AD is the longest bar, then

$$L_{CD} + L_{AD} \leq L_{BC} + L_{AB}, \quad L_{AD} \geq L_{BC}$$

then

$$200 + L_{AD} \leq 300 + 250, \quad L_{AD} \leq 350$$

so that

$$300 \leq L_{AD} \leq 350$$

# HW 02.3

(2) 若  $BC$  为最长杆, 则

$$L_{CD} + L_{BC} \leq L_{AD} + L_{AB}, \quad L_{AD} \leq L_{BC} \quad (5分)$$

即

$$200 + 300 \leq L_{AD} + 250, \quad L_{AD} \leq 300 \quad (3分)$$

即

$$250 \leq L_{AD} \leq 300 \quad (2分)$$

(2) If  $BC$  is the longest bar, then

$$L_{CD} + L_{BC} \leq L_{AD} + L_{AB}, \quad L_{AD} \geq L_{BC}$$

then

$$200 + 300 \leq L_{AD} + 250, \quad L_{AD} \leq 300$$

so that

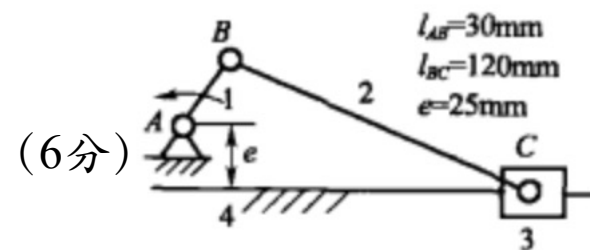
$$250 \leq L_{AD} \leq 300$$

# HW 02.4

在图示机构中

- (1) 以构件1为主动件，机构是否会出现死点位置？如果有，请画出机构的死点位置并表明机构的主动件是哪一个构件
- (2) 以构件3为主动件，机构是否会出现死点位置？如果有，请画出机构的死点位置并表明机构的主动件是哪一个构件

解：(1) 当以构件1为主动件时，机构不会出现死点位置



Solution: (1) When member 1 is the active member, the mechanism does not have a dead position

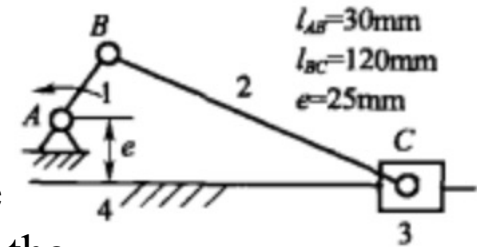
# HW 02.4

在图示机构中

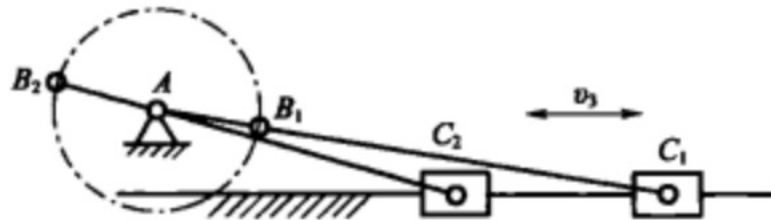
- (1) 以构件1为主动件，机构是否会出现死点位置？如果有，请画出机构的死点位置并表明机构的主动件是哪一个构件
- (2) 以构件3为主动件，机构是否会出现死点位置？如果有，请画出机构的死点位置并表明机构的主动件是哪一个构件

解：(2) 当以构件2为主动件时，机构会出现死点位置，其死点位置如下图所示

(6分)



Solution: (2) When member 2 is used as the active member, the mechanism will have a dead center position, which is shown in the following figure



(8分)



## HW 02.5

(1) 当行程速比系数  $K=1$  时, 机构的极位夹角为

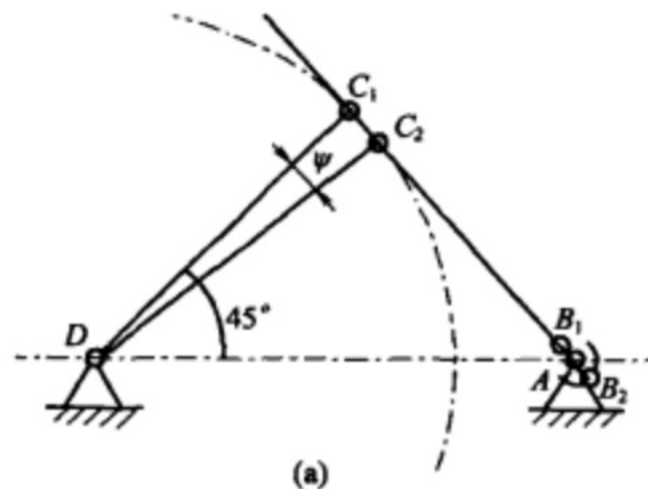
$$\theta = 180^\circ \frac{K-1}{K+1} = 0^\circ \quad (5\text{分})$$

机构没有急回特性, 固定铰链点  $A$  应在活动铰链点  $C$  的两个极限位置  $C_1$ 、 $C_2$  的连线上, 确定活动铰链点  $C$  的另一个极限位置。选定比例尺, 作图 4-15 (a)。直接由图中量取, 所以构件  $AB$  的长为

$$l_{AB} = \frac{\overline{AC_1} - \overline{AC_2}}{2} = \frac{70.84 - 61.76}{2} = 4.54(\text{mm}) \quad (5\text{分})$$

$$l_{BC} = \frac{\overline{AC_1} + \overline{AC_2}}{2} = \frac{70.84 + 61.76}{2} = 66.3(\text{mm}) \quad (5\text{分})$$

$$\psi = 7^\circ \quad (5\text{分})$$

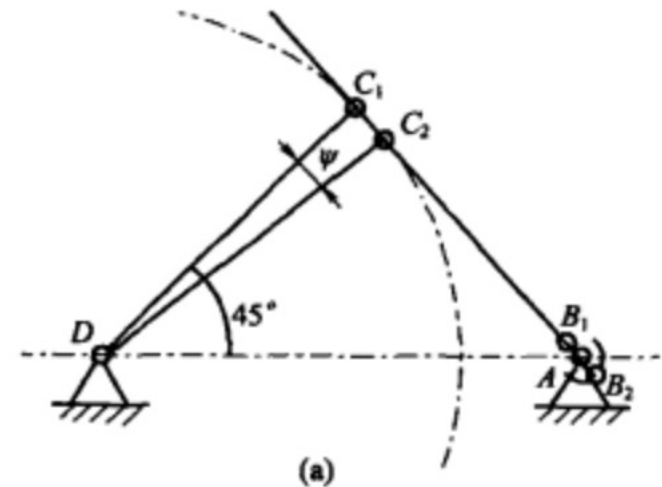


# HW 02.5

When the travel speed ratio coefficient  $K = 1$ , the polar clamping angle of the mechanism is:

$$\theta \approx 180^\circ \frac{K - 1}{K + 1} \approx 0^\circ$$

The mechanism does not have a sharp return property, the fixed hinge point A should be on the line connecting the two limit positions  $C_1$  and  $C_2$  of the movable hinge point C. Determine the other limit position of the movable hinge point C, as shown in Fig.



$$l_{AB} = \frac{AC_1 - AC_2}{2} = \frac{70.84 - 61.76}{2} = 4.54(\text{mm}) \quad (5\text{分})$$

$$l_{BC} = \frac{AC_1 + AC_2}{2} = \frac{70.84 + 61.76}{2} = 66.3(\text{mm}) \quad (5\text{分})$$

$$\psi = 7^\circ \quad (5\text{分})$$